

跨音速平面叶栅全位势无粘-粘性迭代计算

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摘 要

本文发展了一种全位势与附面层、尾迹流正逆模式积分法迭代计算程序,给出了计算结果与国内外四个不同的叶栅风洞试验结果的对比分析。结果表明,对比是较为满意的,本文在网格生成,无粘跨音流求解和无粘流与附面层、尾迹流迭代等方面所作的发展和改进适用,使整个程序系统具有工程实用价值。

一、引 言

把整个流场(包括位势流区和边界层区)统一用NS方程计算,由于流场不同区域内尺度相差很大,网格设计有困难,计算机速度和内存要求也过高。同时,由于紊流模型还不成熟。因而,这种算法还不适合工程应用。而无粘与附面层迭代解所用的方程较简单,计算时间和内存可显著减少。且由于附面层目前已突破薄层近似的概念,无论从计算技术上还是物理模型上都已取得许多进展,无粘与附面层迭代正朝着间接求解NS方程的方向发展。

在叶栅内流中,跨音流态比外流更复杂。例如,叶栅中由于存在强逆压梯度,附面层易于分离,叶栅尾迹模型也还需做工作。因此,叶栅中无粘与附面层迭代的工作还不多。文献〔1〕采用时间相关法解欧拉方程与附面层正逆模式积分法迭代,得到较好结果,但该文处理尾迹存在不足。文献〔2〕采用渗透壁法处理叶面附面层及尾迹,在最佳攻角附近得到可与实验相比较的结果,但渗透壁法在叶栅内流中计算较复杂,尤其对AF2分裂格式更为困难。

本文给出一种无粘与附面层迭代计算方案,其程序处理有如下特点:

- 1 由于本文作者发展了快速几何方法来生成网格〔3〕,无粘与附面层迭代计算时只需少量变动等 η 网格线,耗费机时少且简便。
- 2 由于无粘流场采用AF2快速分裂算法〔3〕,使无粘与附面层迭代耗费机时保持在实用允许的范围。
- 3 本文应用库塔条件把出气角作为解的一部分迭代算出,同时严格保证叶栅进出口以及各纵向计算站之间流量相等。这些措施使得准确预估落后角成为可能。

二、网格生成与全位势方程AF2格式

文献〔3〕提出一种叶栅H网格几何构造法,并对网格参数的影响作了研究。给出了全位势AF2格式计算跨音叶栅的程序,改进了人工粘性格式。对较大弯度和厚度跨音叶栅,得到

了与速度图法与复速度图法同等精度的计算结果。

AF2差分格式为〔8〕:

$$(\alpha - \delta_z^* (\tilde{P} A_1/J)_{i-1/2, j}) (\alpha \delta_z^* - \delta_n^* (\rho A_3/J)_{i, j+1} \delta_n^*) C_{1, j}^n = \alpha \omega L \phi_{1, j}^n$$

三、附面层与尾迹计算

附面层计算可分为三个部分: 层流、转捩及紊流附面层和尾迹流。

层流附面层的计算采用 Thwaites 法〔4〕, 转捩计算采用 Dunham 经验方法〔5〕, 紊流附面层和尾迹流的计算采用文献〔6〕的延迟裹入法。

目前, 由于实验数据范围的限制, 已有的形状因子经验关系还不完全适用于有大分离泡的流动, 仅适用于有小分离泡流动的计算。

本文采用上述方法, 用 Runge-Kutta 积分法编制了计算程序。为检验其精度, 用 Korn 超临界叶栅附面层数据进行了验算, 结果对比见图 1。由图可知, 本程序计算精度与较复杂的附面层程序〔7〕相同。

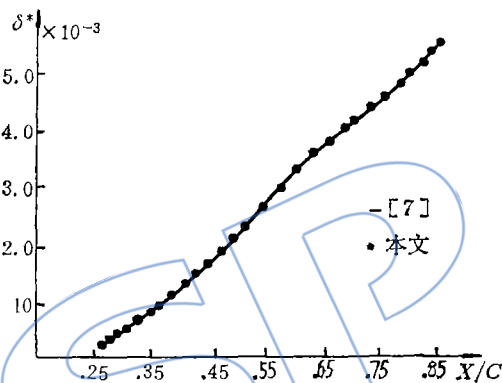


图1 Korn叶栅附面层位移厚度验算

四、无粘流与附面层迭代

为简单起见, 用附面层外缘速度 U_e 与位移厚度 δ^* 的控制方程的缩写形式表示。

流场中无粘与附面层粘性区域的划分, 使每一部分产生一种 U_e 与 δ^* 之间的关系:

无粘流: $U_e = P(\delta^*)$

粘性流: $U_e = Q(\delta^*)$

经典迭代法可写作:

正模式: $U_e^{(n)} = P(\delta^{*(n-1)}) \quad \delta^{*(n)} = Q^{-1}(U_e^{(n)})$

接近分离点时, Q^{-1} 具有 Goldstein 奇异性, 正模式积分遇到困难, 可由逆模式积分避开。

逆模式: $\delta^{*(n)} = P^{-1}(U_e^{(n-1)}) \quad U_e^{(n)} = Q(\delta^{*(n)})$

但这种迭代收敛较慢。较快的方法是以上两者的结合, 即所谓半逆模式:

$U_e^P = P(\delta^{*(n-1)}) \quad U_e^B = Q(\delta^{*(n-1)})$

$\delta^{*(n)} = R(U_e^P, U_e^B, \delta^{*(n-1)})$

这里 R 是松弛公式。

本文采用无粘与附面层、尾迹流正模式及半逆模式迭代法, 当 $\bar{H}$ 达 2.0 时, 即接近分离点时, 采用半逆模式。

无粘流与尾迹流之间的迭代比叶型附面层与无粘流之间的迭代更复杂。目前常采用简化处理。如著名的超临界叶栅, 用尾迹中心线为直线及尾迹两侧静压相等的假设, 得到了与风

洞试验相比相当满意的结果^[8]。文献^[9]对两套压气机叶栅的尾迹特性作了详细的实验研究。结果表明,除了在尾缘邻近处稍有差别外,大部分尾迹两侧静压是相等的。由于所试验的两套叶栅尾缘都有相当大的圆角,尾缘处存在看旋涡,测量数据在该小区域内有一定分散性,在此小区域内还得出不了定量的实验规律。文献^[1、2]均采用全部尾迹两侧静压相等的假设,得到了好的计算结果。

考虑到上述实验及数值计算结果,以及本文采用的附面层及尾迹计算方法还不适用于大分离流动(因而不能计及法向压力梯度的影响),所以仍采用尾迹两侧静压相等的假设,并按这一要求对尾迹中心线形状进行调整。结果表明尾迹中心线曲率很小,得到与文献^[2]相同的结果。同时还表明绝大部分尾迹两侧静压相等,仅在尾缘一个小区域内,尾迹中心线不论如何调整,两侧静压差只能达到某一最小值,而不能达到零。这反映出该小区域内尾迹中心线曲率及附面层法向压力梯度的影响已不能忽略不计。对此,今后需进一步做工作。

五、计算结果与实验的对比分析

为检验本文发展的跨音叶栅流场预测系统的精度,采用国内外四套叶栅实验数据作了对比分析。

1 北京航空学院C₄叶栅^[10]

叶型弯角 $\theta = 25^\circ$, $Ma_1 = 0.30$, 当攻角 $i = -0.9^\circ$ 时, 计算结果与试验对比见图 2。

由图可见, 计算时如取密流比 $K = 1.0$, 则叶面压力分布普遍向下平移一个距离, 还能与试验符合。由于试验测得密流比 $K = 0.922$, 考虑密流比修正后, 计算与试验符合良好。密流比修正时, 取栅前区为1.0, 栅后区为0.922, 叶栅槽道中线性分布。

2 冯·卡门研究院双圆弧叶栅^[11]

叶型弯角 $\theta = 9.5^\circ$, $Ma_1 = 0.70$, 攻角 $i = 0.6^\circ$ 时, 实验测得密流比 $K = 0.997$, 故计算时未作密流比修正。

计算结果见图 3 到图 7。由图 3 可知, 计算与实验对比是满意的。

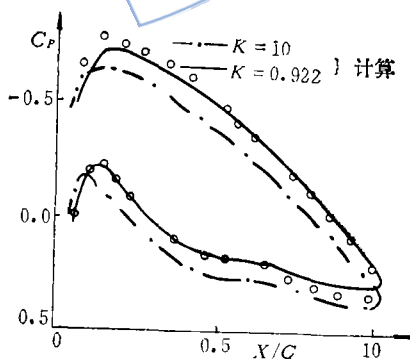


图 2 $i = -0.9^\circ$ 时C₄叶栅面压力分布

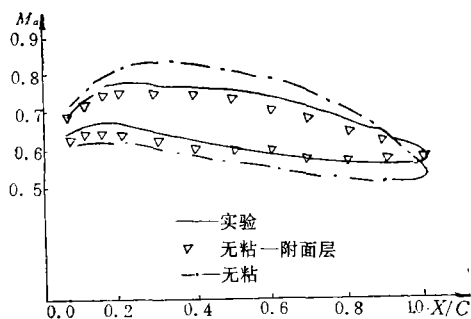
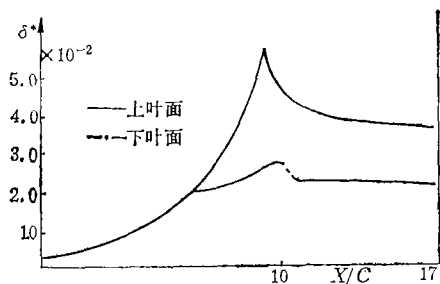
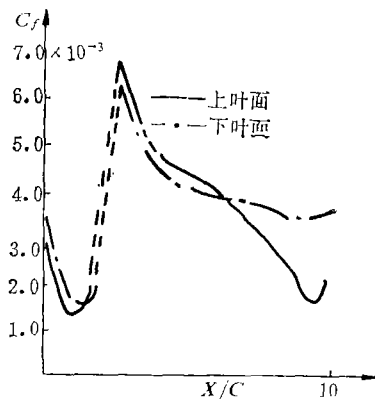
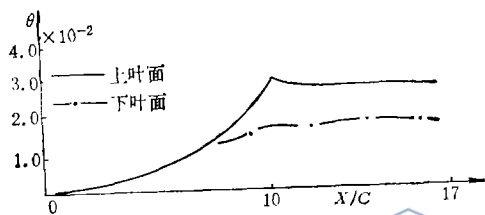
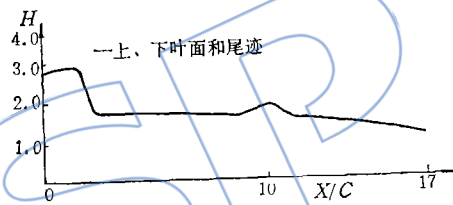


图 3 叶面马赫数计算与实验对比

图 4 位移厚度 δ^* 沿叶面和尾迹分布图 5 阻力系数 C_f 沿叶面分布图 6 动量厚度 θ 沿叶面和尾迹分布图 7 形状因子 H 沿叶面和尾迹分布表 1 损失系数 $\bar{\omega}$ 和落后角 δ 对比

	损 失 系 数 $\bar{\omega}$	落 后 角 δ
实 验	2.50%	2.5°
计 算	2.46%	2.03°

3 NAL双圆弧叶栅^[12]

叶型弯角 $\theta = 11.2^\circ$ ，攻角 $i = -3^\circ$ ，计算了四个来流马赫数工况，所得结果与实验对比见图 8 和图 9。

由图 8、9 可见，计算与实验符合程度是满意的。当来流马赫数超过临界马赫数过多时，损失急剧升高，附面层有严重分离，目前本文还无力处理。

4 燃气涡轮研究所多圆弧叶栅

叶型弯角 $\theta = 7.8^\circ$ ，来流马赫数 $Ma_1 = 0.777$ 时计算与实验对比见图 10。由图可见，在跨音速时，计算与实验大致相符，但存在一定误差。其原因从计算角度看，本叶栅安装角仅 34.3° ，因而使计算网格严重偏离正交，计算误差增大。从实验方面看，风洞出口与计算条件有一定差别。因而，计算与实验的差别，需进一步研究。

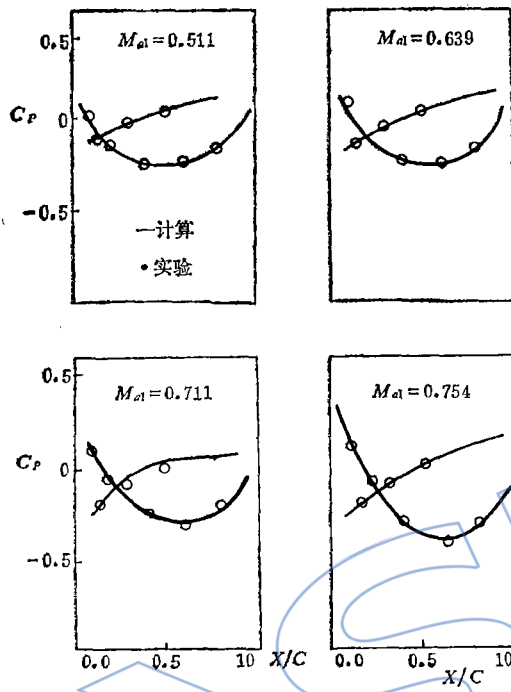


图 8 叶栅叶面压力分布与实验对比

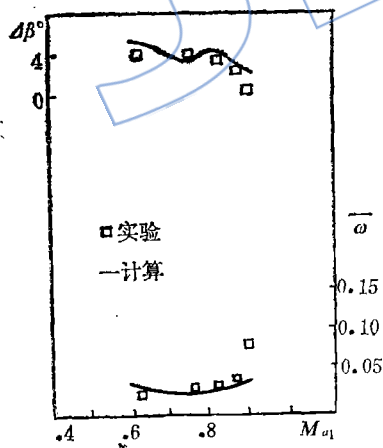
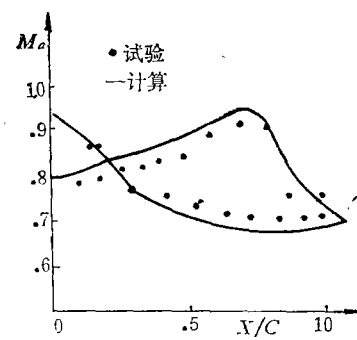
图 9 叶栅损失系数 $\bar{\omega}$ 和转折角 $\Delta\beta$ 与实验的对比

图10 叶面马赫数计算与试验的对比

六、结 论

经过与国内外四个不同叶栅风洞试验数据的比较与分析,可以认为本文发展的无粘流与附面层、尾迹流正-半逆模式积分迭代程序在流动无大分离时,能得到与实验对比较满意的结果。

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PROPAGATION OF COMBINED STEADY CIRCUMFERENTIAL TOTAL PRESSURE AND TEMPERATURE DISTORTION THROUGH AN AXIAL ROTOR

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Abstract

Although the effects of combined steady circumferential total pressure and temperature distortion on performance have been investigated analytically and experimentally in recent years, the other urgent problem concerning the interrelation between them in the process of propagation through an axial compressor (including multistage and two-spool axial compressor) has been somewhat neglected.

On the basis of previous work (4) and the approach proposed by the authors in dealing with the distortion propagation, two cases of combined steady circumferential total pressure and temperature distortion are investigated: first, how the variation of the total temperature distortion amplitude influences the attenuation of the total pressure distortion under the conditions of no phase difference; second, how it does when there is phase difference between them.

Numerical experiment and analysis show that the amplitude of the total temperature distortion and the phase angle between them have great influences upon the attenuation of the total pressure distortion. Within a certain range of the total temperature amplitude, the existence of the total temperature distortion is expedient to the attenuation of the total pressure distortion. The worst case is when the two are just opposite in phase and the total temperature distortion amplifies the total pressure distortion instead of attenuating it.

A METHOD FOR COMPUTATION OF VISCID/INVISCID INTERACTION ON TRANSONIC COMPRESSOR CASCADES

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Abstract

An inviscid/viscous interaction method has been developed to predict the blade-to-blade flow in transonic compressor cascades. The method intends to be used primarily for relatively high camber blades where the suction surface bound-

dary layer may be slightly separated near the trailing edge. The inviscid flow was calculated by means of AF2 finite-difference scheme on a newly constructed H-type grid system[3], and the viscous flow by the aid of an integral method for laminar and turbulent boundary layers[6]. An integration calculation was then carried out to determine the blade deviation angle and the total pressure loss coefficient. One of the authors has been involved in the recent transonic experimental work of one set of MCA cascades in Sichuan. The numerical predictions obtained have been compared with test data on four different transonic cascades in reasonably good agreement.

OPTIMUM COMPUTATION FOR EXIT CONE CONTOUR OF NOZZLE WITH TWO-PHASE FLOW

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Abstract

To simplify the computational procedure and minimize the expensive computer time, the optimum calculation for exit cone contour of nozzle with two-phase flow is carried out by the procedure of two-dimensional two-phase (2D-2-phase) nozzle flow. This procedure consists of computations of both transonic and supersonic two-phase flow. The former provides an exact data at the initial line for the latter.

The computation of supersonic two-phase flow is completed by characteristics method. The direct marching method is applied to the interior points and axis points, and the inverse marching method to the wall points. There are seventeen finite difference grids in the flowfield, but the typical grids are only five, the others can be divided into several typical grids.

The influences of inertia, compression and rarefaction must be considered in the particle drag and heat transfer formulae for computing the supersonic two-phase flow.

The 2D-2-phase specific impulse losses have been computed by the present procedure, and the results illustrate that the optimum initial expansion angles are about 28.5° , 29.5° , and 27.0° for parabolic, circular and cubic contours respectively. The 2D-2-phase specific impulse losses of these three nozzle configurations are almost the same when the exit angles are equal and the initial angles are close for the above curves.